

CBE 60545

Transport Phenomena II

Course Notes

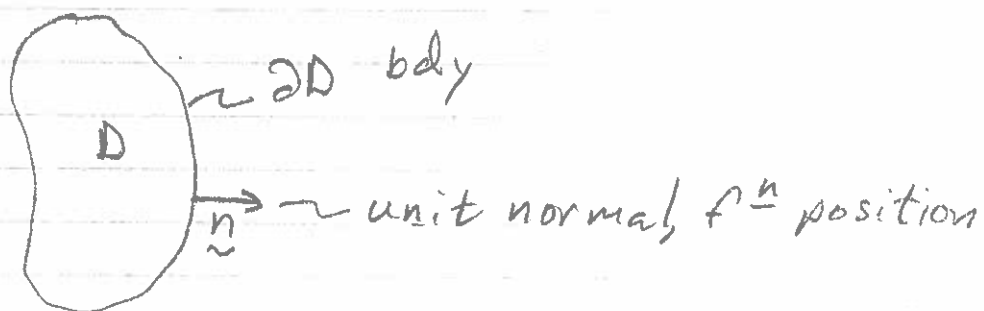
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Deriv. Transport Eqns:

Review mat'l in ch 8 BSD L $\Rightarrow$ don't memorize, but know it exists & how to use. Deals w/ ~~Area~~ Law & thermal conductivity fourier law of ht. cond.

Consider arbitrary volume D :



We have some fluid flowing thru volume

Derive transp. eqns from conservation laws:

Mass is conserved, thus

$$\left\{ \begin{array}{l} \text{rate of} \\ \text{accumulation} \end{array} \right\} = - \left\{ \begin{array}{l} \text{net rate outflow} \\ \text{by convection} \end{array} \right\}$$

Now rate of accumulation is simply

$$\{ \text{acc.} \} = \frac{d}{dt} \int_D \rho \, dV = \int_D \frac{\partial \rho}{\partial t} \, dV$$

And rate of outflow must occur at surface ∂D . Flux thru any surface is just

$$\text{Flux} = \rho(\underline{u} \cdot \underline{n}) \sim \text{component of velocity in dir. of unit normal}$$

Thus

$$\left\{ \begin{array}{l} \text{net rate outflow} \\ \text{by convection} \end{array} \right\} = \int_{\partial D} \rho \underline{u} \cdot \underline{n} \, dA$$

$$\text{or } \int_D \frac{\partial \rho}{\partial t} \, dV + \int_{\partial D} \rho \underline{u} \cdot \underline{n} \, dA = 0$$

Now use Divergence Theorem:

$$\int_{\partial D} \underline{f} \cdot \underline{n} \, dA = \int_D \underline{\nabla} \cdot \underline{f} \, dV$$

Thus

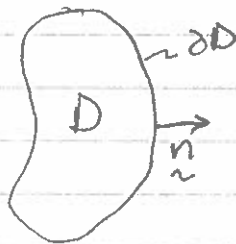
$$\int_D \left[\frac{\partial \rho}{\partial t} + \underline{\nabla} \cdot (\rho \underline{u}) \right] dV = 0$$

Since D is arbitrary, eq'n valid at all pts, thus:

$$\frac{\partial \rho}{\partial t} + \underline{\nabla} \cdot (\rho \underline{u}) = 0$$

which is eqn of continuity. Will use same mass balance for multicomponent case to derive mass transp. eqns later.

Now look at energy: also conserved quantity!



$$\begin{aligned}
 \left\{ \begin{array}{l} \text{Rate of} \\ \text{accumulation} \\ \text{of energy} \end{array} \right\} &= - \left\{ \begin{array}{l} \text{net rate}^{\text{of energy}} \text{ out} \\ \text{by convection} \end{array} \right\} \\
 - \left\{ \begin{array}{l} \text{net rate}^{\text{out}} \\ \text{by conduction} \end{array} \right\} &- \left\{ \begin{array}{l} \text{net rate work by} \\ \text{system on surroundings} \end{array} \right\} \\
 &\quad \text{(we include gravity work here)} \\
 + \left\{ \begin{array}{l} \text{sources} \end{array} \right\} & \\
 &\quad \searrow \text{ignore for now}
 \end{aligned}$$

The energy of fluid element is just internal energy + K.E., thus

$$\text{Tot. E. per unit volume} \equiv \rho \hat{U} + \frac{1}{2} \rho u^2$$

$\hat{U}$ → int. E / unit mass
 ρ → pot. E included as work term

Thus:

$$\left\{ \begin{array}{l} \text{Rate of} \\ \text{acc. of E} \end{array} \right\} = \int_D \frac{\partial}{\partial t} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) dV$$

and

$$\left\{ \begin{array}{l} \text{net rate E out} \\ \text{by convection} \end{array} \right\} = \int_{\partial D} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \underline{u} \cdot \underline{n} dA$$

let $\underline{q}$ be conduction flux vector,
then

$$\left\{ \begin{array}{l} \text{energy out by} \\ \text{conduction} \end{array} \right\} = \int_{\partial D} \underline{q} \cdot \underline{n} dA$$

Left w/ work terms.

Suppose surroundings exert force/Area $\underline{f}$ on system at surface ∂D

$\therefore$ ^{rate of} work/area done by system on surroundings

$$= -\underline{u} \cdot \underline{f}$$

what is $\underline{f}$?

related to stress tensor $\underline{\sigma}$

(6)

$\sigma_{ij} \equiv F/A$ exerted by fluid of greater i on fluid of lesser i in j direction

Thus, $\underline{f} = \underline{\sigma} \cdot \underline{n}$!

identity matrix

Now recall that $\underline{\sigma} = -p \underline{I} + \underline{\tau}$
 (in Einstein notation, $\sigma_{ij} = -p \delta_{ij} + \tau_{ij}$)
 $\rightarrow$ dynamic or viscous stress

$$\sigma_{ij} = -p \delta_{ij} + \tau_{ij}$$

Note: BS&L uses opposite sign for $\tau_{ij} \Rightarrow$ don't worry about it, just keep definition straight.

So: work/Area by system on surroundings is:

$$\frac{W}{A} = -\underline{u} \cdot \underline{f} = -\underline{u} \cdot (\underline{\sigma} \cdot \underline{n})$$

$$= -\underline{u} \cdot \left(-p \underline{I} + \underline{\tau} \right) \cdot \underline{n}$$

$$= +p \underline{u} \cdot \underline{n} - \underline{u} \cdot \underline{\tau} \cdot \underline{n}$$

$\uparrow$
pressure
work

$\uparrow$
viscous
work

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We also have work done against gravity, given by:

$$\left\{ \begin{array}{l} \text{rate of work done} \\ \text{by gravity on} \\ \text{system} \end{array} \right\} = \int_D \rho \underline{u} \cdot \underline{g} \, dV$$

↑
source of K.E.

Putting it all together:

$$\begin{aligned} \int_D \frac{\partial}{\partial t} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) dV &= - \int_{\partial D} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \underline{u} \cdot \underline{n} \, dA \\ &\quad - \int_{\partial D} \underline{q} \cdot \underline{n} \, dA - \int_{\partial D} p \underline{u} \cdot \underline{n} \, dA \\ &\quad + \int_{\partial D} \underline{u} \cdot \underline{\tau} \cdot \underline{n} \, dA + \int_D \rho \underline{u} \cdot \underline{g} \, dV \end{aligned}$$

Accumulation

Convection

conduction

pressure work

viscous work

gravity work

Applying the Divergence theorem to RHS:

$$\begin{aligned} \int_D \frac{\partial}{\partial t} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) dV &= - \int_D \nabla \cdot \left[\left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \underline{u} \right] dV \\ &\quad - \int_D \underline{\tau} \cdot \underline{e} dV - \int_D \nabla \cdot (P \underline{u}) dV \\ &\quad + \int_D \nabla \cdot (\underline{\tau} \cdot \underline{u}) dV + \int_D \rho \underline{u} \cdot \underline{g} dV \end{aligned}$$

Or, since D is arbitrary,

$$\begin{aligned} \frac{\partial}{\partial t} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) &= - \nabla \cdot \left[\left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \underline{u} \right] - \nabla \cdot \underline{\tau} \\ &\quad - \nabla \cdot (P \underline{u}) + \nabla \cdot (\underline{\tau} \cdot \underline{u}) + \rho \underline{u} \cdot \underline{g} \end{aligned}$$

Still a bit unweildy.

Now diff. first term on RHS by parts:

$$\begin{aligned} \nabla \cdot \left[\left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \underline{u} \right] &= \rho \underline{u} \cdot \nabla \left(\hat{U} + \frac{1}{2} u^2 \right) \\ &\quad + \left(\hat{U} + \frac{1}{2} u^2 \right) \nabla \cdot (\rho \underline{u}) \end{aligned}$$

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And term on LHS:

$$\left(\frac{\partial}{\partial t} \left(\rho \hat{U} + \frac{1}{2} \rho u^2 \right) \right) = \rho \frac{\partial}{\partial t} \left(\hat{U} + \frac{1}{2} u^2 \right) + \left(\hat{U} + \frac{1}{2} u^2 \right) \frac{\partial \rho}{\partial t}$$

Thus we obtain:

$$\begin{aligned} \rho \left(\frac{\partial}{\partial t} + \underline{u} \cdot \underline{\nabla} \right) \left(\hat{U} + \frac{1}{2} u^2 \right) &\equiv \rho \frac{D}{Dt} \left(\hat{U} + \frac{1}{2} u^2 \right) \\ &= - \left(\hat{U} + \frac{1}{2} u^2 \right) \left[\frac{\partial \rho}{\partial t} + \underline{\nabla} \cdot (\rho \underline{u}) \right] \\ &\quad \downarrow \text{from C.E.} \end{aligned}$$

$$= \underline{\nabla} \cdot \underline{q} + \rho \underline{u} \cdot \underline{g} - \underline{\nabla} \cdot (P \underline{u}) + \underline{\nabla} \cdot (\underline{\tau} \cdot \underline{u})$$

which is the equation of energy from viewpoint of an observer moving with the fluid $\Rightarrow$ Lagrangian viewpoint.

Still a bit unweirdy $\Rightarrow$ remove mech. E terms.

Mech. Energy Balance $\Rightarrow$ derivable from M-S eqns.

Recall:

$$\rho \frac{D \underline{u}}{Dt} = - \underline{\nabla} P + \rho \underline{g} + \underline{\nabla} \cdot \underline{\tau}$$

note sign dif. from BSL

Dot w/ velocity & obtain:

$$\rho \frac{D(\frac{1}{2}u^2)}{Dt} = -\underbrace{\underline{u} \cdot \underline{\nabla} P}_{\downarrow} + \underbrace{(\underline{u} \cdot (\underline{\nabla} \cdot \underline{\tau}))}_{\downarrow} + \rho \underline{u} \cdot \underline{g}$$

$$= \rho(\underline{\nabla} \cdot \underline{u}) - \underline{\nabla} \cdot (P\underline{u}) \quad \bigg| \quad = \underline{\nabla} \cdot (\underline{\tau} \cdot \underline{u}) - \underline{\tau} : \underline{\nabla} \underline{u}$$

Thus we obtain:

$$\rho \frac{D(\frac{u^2}{2})}{Dt} = \underbrace{\rho(\underline{\nabla} \cdot \underline{u}) - \underline{\nabla} \cdot (P\underline{u})}_{\substack{\text{Total pressure work} \\ \hookrightarrow \text{Rev. conv. of KE to IE}}} + \underbrace{\underline{\nabla} \cdot (\underline{\tau} \cdot \underline{u})}_{\substack{\text{total viscous work} \\ \uparrow}} - \underline{\tau} : \underline{\nabla} \underline{u}$$

$\hookrightarrow \text{irrev. conversion of mech. E to thermal E}$

Subtraction from total E eq'n yields:

$$\rho \frac{D\hat{U}}{Dt} = -\underline{\nabla} \cdot \underline{q} - P(\underline{\nabla} \cdot \underline{u}) + \underline{\tau} : \underline{\nabla} \underline{u}$$

$\hookrightarrow$ may be pos. or neg.

End lec. 1 Thermal energy eq'n

More convenient to work w/ eq'n in terms of temp.

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From thermo, assuming no changes in composition,

$$U = f^{\wedge}(\hat{V}, T)$$

$$\therefore d\hat{U} = \left(\frac{\partial \hat{U}}{\partial \hat{V}} \right)_T d\hat{V} + \left(\frac{\partial \hat{U}}{\partial T} \right)_{\hat{V}} dT$$

$\hookrightarrow \hat{C}_V dT$

$$\rightarrow \left[-P + T \left(\frac{\partial P}{\partial T} \right)_{\hat{V}} \right] d\hat{V}$$

from thermodynamics

Thus

$$\rho \frac{D\hat{U}}{Dt} = \left[-P + T \left(\frac{\partial P}{\partial T} \right)_{\hat{V}} \right] \rho \frac{D\hat{V}}{Dt} + \rho \hat{C}_V \frac{DT}{Dt}$$

Now

$$\rho \frac{D\hat{V}}{Dt} = \rho \frac{D(1/\rho)}{Dt} = -\frac{1}{\rho} \frac{D\rho}{Dt} = \nabla \cdot \underline{u}$$

from continuity

Thus

$$\rho \frac{D\hat{U}}{Dt} = \left[-P + T \left(\frac{\partial P}{\partial T} \right)_{\hat{V}} \right] (\nabla \cdot \underline{u}) + \rho \hat{C}_V \frac{DT}{Dt}$$

and thus

$$\rho \hat{C}_V \frac{DT}{Dt} = \underbrace{-\nabla \cdot \underline{q}}_{\text{conduction}} - \underbrace{T \left(\frac{\partial P}{\partial T} \right)_{\hat{V}} \nabla \cdot \underline{u}}_{\text{compression}} + \underbrace{\underline{\tau} : \nabla \underline{u}}_{\text{viscous dissipation}}$$

This form is useful when dealing with gases:

For an ideal gas $PV = RT$

$$\therefore T \left(\frac{\partial P}{\partial T} \right)_V = \frac{TR}{V} = P$$

Also, for a Newtonian fluid

$$\tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} - \frac{2}{3} \nabla \cdot \underline{u} \delta_{ij} \right)$$

thus

$$\underline{\tau} : \underline{\nabla u} \equiv \mu \phi_v = \tau_{ij} \frac{\partial u_i}{\partial x_j} = \mu \frac{1}{2} \left[\left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)^2 - \frac{4}{3} (\nabla \cdot \underline{u})^2 \right]$$

positive
definite!

And from Fourier Law of ht. conduction:

$$\underline{q} = -k \underline{\nabla T}$$

↑
thermal conductivity

Thus

$$\rho \hat{C}_v \frac{DT}{Dt} = \underbrace{\nabla \cdot (k \underline{\nabla T})}_{\text{conduction}} - \underbrace{P \nabla \cdot \underline{u}}_{\text{adiabatic compression}} + \mu \phi_v$$

→ an example of where this would be used is in calc. of temp. distrib. in a shock tube

this is starting point for energy transp. in gases → note ϕ usually negligible
 at cst. vol.

For liquids & solids, or gases at cst. pressure, a more convenient form may be used

Return to thermal E eq'n:

$$\rho \frac{D\hat{U}}{Dt} = -\nabla \cdot \underline{q} - P(\nabla \cdot \underline{u}) + \underline{\tau} : \nabla \underline{u}$$

Recall from thermo: $\hat{H} = \hat{U} + P\hat{V} = \hat{U} + \frac{P}{\rho}$

Thus

$$\begin{aligned} \frac{D\hat{H}}{Dt} &= \frac{D\hat{U}}{Dt} + \frac{1}{\rho} \frac{DP}{Dt} + P \frac{D(1/\rho)}{Dt} \\ &= \frac{D\hat{U}}{Dt} + \frac{1}{\rho} \frac{DP}{Dt} - \frac{P}{\rho^2} \left(\frac{D\rho}{Dt} \right) \rightarrow -\rho \nabla \cdot \underline{u} \text{ from C.E.} \end{aligned}$$

Thus $\rho \frac{D\hat{U}}{Dt} = \rho \frac{D\hat{H}}{Dt} - \frac{DP}{Dt} + P \nabla \cdot \underline{u}$

And from internal energy eq'n:

$$\rho \frac{D\hat{H}}{Dt} = -\nabla \cdot \underline{q} + \frac{DP}{Dt} + \underline{\tau} : \nabla \underline{u}$$

Now, again from thermodynamics

$$\begin{aligned} d\hat{H} &= \left(\frac{\partial \hat{H}}{\partial T} \right)_P dT + \left(\frac{\partial \hat{H}}{\partial P} \right)_T dP \\ &= \hat{C}_P dT + \left[\hat{V} - T \left(\frac{\partial \hat{V}}{\partial T} \right)_P \right] dP \end{aligned}$$

thus

$$\rho \frac{D\hat{H}}{Dt} = \rho \hat{C}_P \frac{DT}{Dt} + \frac{DP}{Dt} - \beta T \frac{DP}{Dt}$$

where $\beta \equiv \frac{1}{\hat{V}} \left(\frac{\partial \hat{V}}{\partial T} \right)_P$ = thermal expansion

thus

$$\rho \hat{C}_P \frac{DT}{Dt} = -\nabla \cdot \underline{q} + \beta T \frac{DP}{Dt} + \underline{\tau} : \underline{\nabla} \underline{u}$$

Now if we are at cst. pressure then

$$\frac{DP}{Dt} = 0 \text{ and}$$

$$\rho \hat{C}_P \frac{DT}{Dt} = -\nabla \cdot \underline{q} + \underline{\tau} : \underline{\nabla} \underline{u}$$

or, for cst k & Newtonian fluids:

$$\rho \hat{C}_P \frac{DT}{Dt} = k \nabla^2 T + \mu \phi_v \quad \begin{matrix} \swarrow \\ \text{small in most} \\ \text{cases} \end{matrix}$$

for incompressible fluids, $\hat{C}_P = \hat{C}_V$ and both equations are the same.

For solids $u = 0$ and we obtain

$$\rho \hat{C}_p \frac{\partial T}{\partial t} = K \nabla^2 T \quad (\text{const } K)$$

May also have source terms added to RHS such as electrical dissip, nuclear generation, etc.

would be $E/\text{time}/\text{volume}$

Suppose we have some general problem, say flow past nuclear element of strange shape. Equations governing problem are:

$$\text{C.E.: } \frac{\partial s}{\partial t} + \nabla \cdot (s \underline{u}) = 0 \quad (1)$$

$$\text{N-S: } \rho \frac{D \underline{u}}{Dt} = -\nabla p + (\nabla \cdot \underline{\underline{\tau}}) + \rho \underline{g} \quad (3)$$

$$\text{Thermal: } \rho \hat{C}_p \frac{DT}{Dt} = -(\nabla \cdot \underline{q}) + \beta T \frac{DP}{Dt} + (\underline{\underline{\tau}} : \nabla \underline{u}) + \dot{s}_{th} \quad (1)$$

we also must have constitutive relation:

$$\underline{q} = f^n(K, T); \quad \underline{\underline{\tau}} = f^n(\mu, \underline{u})$$

$$\text{and } \mu, \rho, K = f^n(T, P)$$

Thus, obtain series of coupled equations, must be solved w/ appropriate B.C.'s & I.C.'s

Very difficult to solve (even numerically)

All physics (except radiative ht. transf, ~~not covered in course~~) included in eqns $\Rightarrow$ simply a matter of solution! Physicist would say stop here, as eng. need sol'n $\Rightarrow$ make approximations to obtain sol'n

Problem: apply physical insight to see what approx. are realistic.

OK, now look at energy transp. in solids

we have:

$$\rho \hat{C}_p \frac{DT}{Dt} = -(\nabla \cdot \underline{\underline{q}}) + \left(\frac{\partial \ln \hat{V}}{\partial \ln T} \right)_p \frac{DP}{Dt} + (\underline{\underline{\tau}} : \nabla \underline{\underline{u}}) + \dot{S}_{th}$$

Assume $\frac{DP}{Dt} = 0$ (term is small anyway)

also $\underline{\underline{u}} = 0$ in solid

Substituting Fourier Law of ht cond, obtain

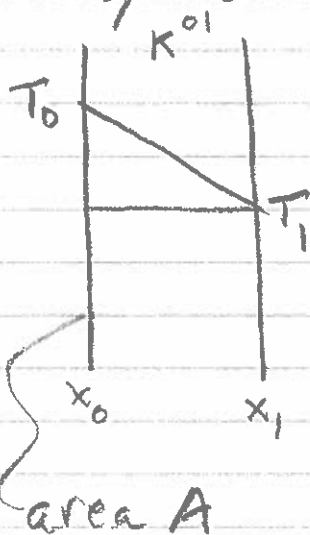
$$\rho \hat{C}_p \frac{\partial T}{\partial t} = \nabla \cdot (\underline{\underline{K}} \nabla T) + \dot{S}_{th}$$

+ relations of $\rho, \hat{C}_p, \underline{\underline{K}}$ to T

Assume $\underline{\underline{K}} = \text{cst}$

$$\therefore \rho C_p \frac{\partial T}{\partial t} = k \nabla^2 T + \dot{S}_{th}$$

Now look at conduction thru wall at steady state (insulation problem)



$$S.S., \text{ so } \frac{\partial T}{\partial t} = 0$$

Given that outside wall is at T_1 , inside wall at T_0 , what will be $\dot{E}_{loss}$?

(what will be heating cost?)

$$\nabla^2 T = 0, \text{ only 1D so } \frac{\partial^2 T}{\partial x^2} = 0$$

$$\frac{\partial T}{\partial x} = C_1, \quad T = C_1 x + C_2$$

$$T|_{(x=x_0)} = T_0, \quad T|_{x=x_1} = T_1$$

$$\therefore T = T_0 + \frac{x - x_0}{x_1 - x_0} (T_1 - T_0)$$

linear profile!

As a consequence of geometry, energy flux thru all planes are equal

$$\text{i.e., } \nabla \cdot \underline{q} = 0 \Rightarrow \frac{\partial q_x}{\partial x} = 0 \therefore q_x = \text{cst}$$

$$q_x = -k \frac{\partial T}{\partial x} = -K \frac{T_1 - T_0}{x_1 - x_0} = -K \frac{\Delta T}{\Delta x}$$

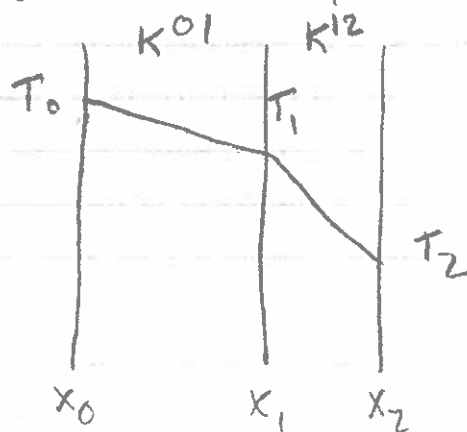
and $\dot{Q} = A q_x \Rightarrow$ total E loss thru wall

Analogy to electrical conduction $\Rightarrow$ voltage drop across resistor exactly equiv. to temp. drop across insulator.

Current equiv. to $\dot{Q}$, $R \sim \frac{\Delta x}{KA}$

$\hookrightarrow$ as shown in BSL ch 8, numerical corresp. as well for metals as both E & elec. carried by electrons $\Rightarrow$ ht trans. resis $\sim$ elec. resis.

Heating bills are too high, so propose to add insulation. Now looks like



still have eqns

$$\frac{\partial^2 T}{\partial x^2} = 0 \text{ in both regions } (q_x = \text{cst})$$

In region I : $T = T_0 + \frac{x - x_0}{x_1 - x_0} (T_1 - T_0)$

II : $T = T_1 + \frac{x - x_0}{x_1 - x_0} (T_2 - T_1)$

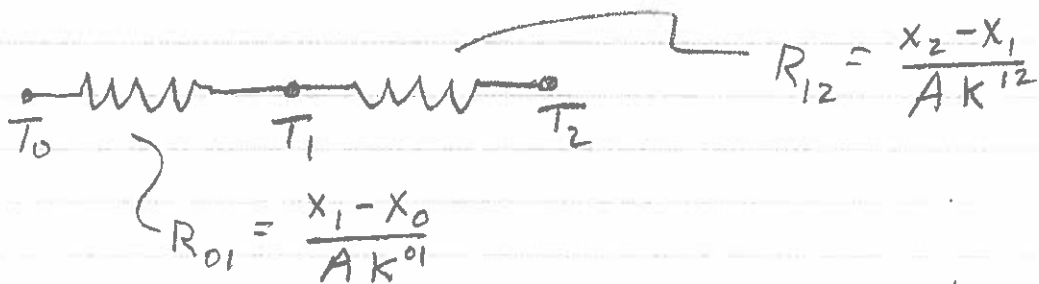
Determine T_1 by condition that $q_x = \text{cst}$ for both regions

$$\therefore -K^{01} \frac{T_1 - T_0}{x_1 - x_0} = -K^{12} \frac{T_2 - T_1}{x_2 - x_1}$$

Thus $T_1 = T_0 + (T_2 - T_0) \left(1 - \frac{K^{12}}{K^{01}} \frac{x_1 - x_0}{x_2 - x_1} \right)$

and $\dot{Q} = (T_2 - T_0) \left(\frac{1}{\frac{x_1 - x_0}{A K^{01}} + \frac{x_2 - x_1}{A K^{12}}} \right)$

Equiv. to conduction in a wire : (series)



Suppose wall was brick - $\therefore K^{01} = 1.5 \times 10^{-3} \text{ cal/sec cm}^2 \text{ } ^\circ\text{K}$
and insulation is $K^{12} = 10^{-4} \text{ cal/sec cm}^2 \text{ } ^\circ\text{K}$

Thus if we have 5" brick & 2" insulation,
reduce cond. losses by:

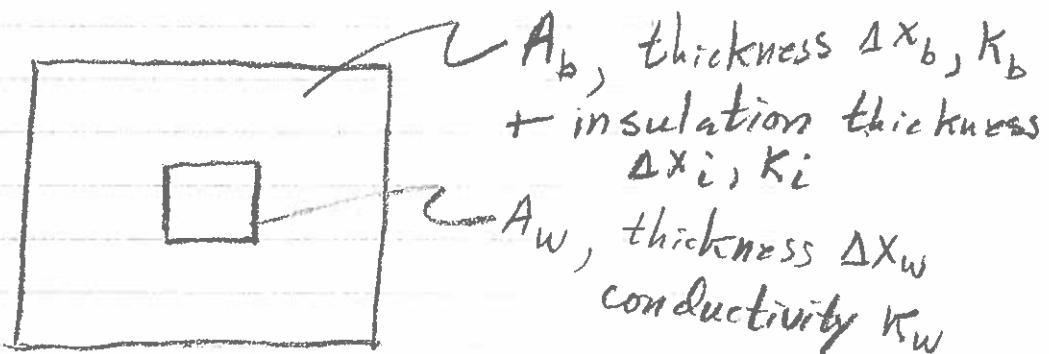
$$\frac{\frac{x_2 - x_1}{A K_{12}}}{\frac{x_2 - x_1}{A K_{12}} + \frac{x_1 - x_0}{A K_{16}}} = .14, \text{ a factor of } 7!$$

Actual calculation more complex as should
include transp. losses thru films, etc.

but shows lge effect of first bit of
insulation.

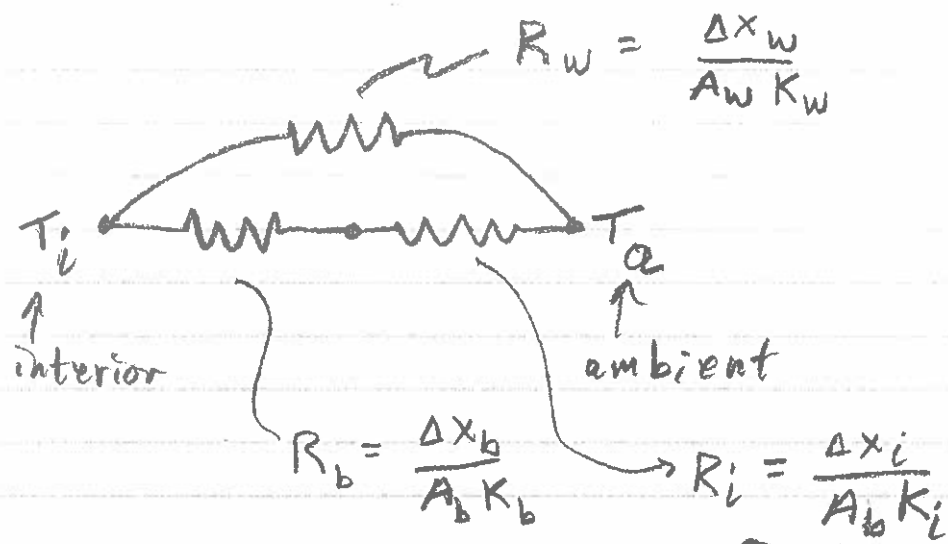
End
Lec. 2

Can similarly work problems for
resistances in parallel: wall w/ window



What is energy loss?

Draw electrical circuit:

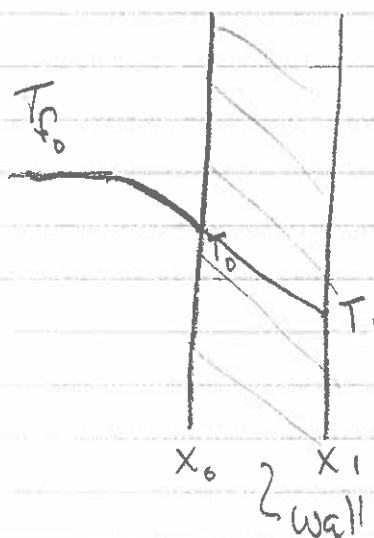


$$\therefore R = \left[\frac{1}{R_b + R_i} + \frac{1}{R_w} \right]^{-1}$$

parallel resistors

$$\text{Thus } \dot{Q} = \frac{(T_i - T_a)}{R}$$

Conduction thru films: ht transf. coef.



Take case of forced convection
 $\therefore$ flow indep. of T profile

As before, ht flux (now both conv. & cond.) thru any plane $x = \text{cst.}$ is same

$\therefore$ Define ht. transf. coef. h s.t.

$$q_x = h (T_f - T_o)$$

for transf. from fluid to surface

Still don't know what h is $\Rightarrow$ calc. is complex, will study later.

Forced conv.: h indep of $(T_f - T_o) \Rightarrow$ prop. to K_f & f^n (flow)

Free conv: $h = f^n (T_f - T_o)$ & sol'n is more difficult

Obtain h from correlations

Solve for total energy transf:

$$\text{in solid, } T = T_o + (T_i - T_o) \frac{x - x_o}{x_i - x_o}$$

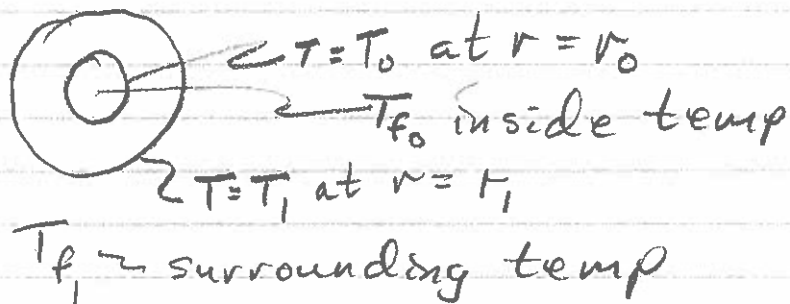
$$\text{and } q_x = \frac{T_i - T_o}{x_i - x_o} K^{oi}$$

$$\text{Thus } Q = (T_f - T_i) \left(\frac{1}{\frac{1}{hA} + \frac{\Delta x_{oi}}{K^{oi}A}} \right)$$

$\Rightarrow$ resistance in series!

Let's look at insulation problem again, only this time in a pipe:

BS&L 9.6-1



Look at energy transp:

S.S., so $\nabla^2 T = 0$

problem symmetric, cylindrical, so use cylindrical coord

$$\frac{d}{dr} \left(r \frac{dT}{dr} \right) = 0 \text{ in insulation}$$

Integrate to get:

$$T = T_0 + \frac{T_1 - T_0}{\ln(r_1/r_0)} \ln\left(\frac{r}{r_1}\right) \text{ in insulation}$$

$$\therefore Q = \frac{T_1 - T_0}{\left(\frac{\ln(r_1/r_0)}{2\pi L K^{01}} \right)} \Leftrightarrow \frac{\Delta T}{R}$$

Assume some h_o at inside wall, h_i at outside

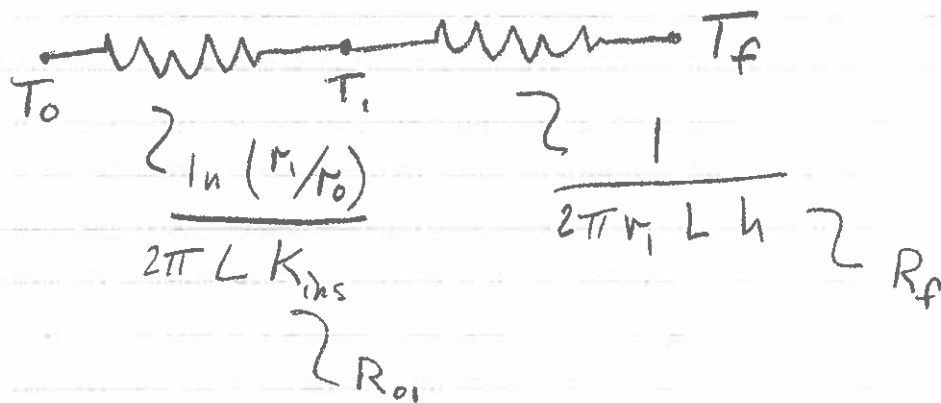
$$Q = \frac{T_{f_o} - T_{f_i}}{\frac{1}{2\pi r_o L h_o} + \frac{\ln(r_i/r_o)}{2\pi L k_{oi}} + \frac{1}{2\pi r_i L h_i}}$$

Since resistances are in series

Critical insulation thickness:

Assume resistance inside $\left(\frac{1}{2\pi r_o L h_o}\right)$ is negligible compared to insulation $\Rightarrow$ good for, say, steam condensing on metal wall.

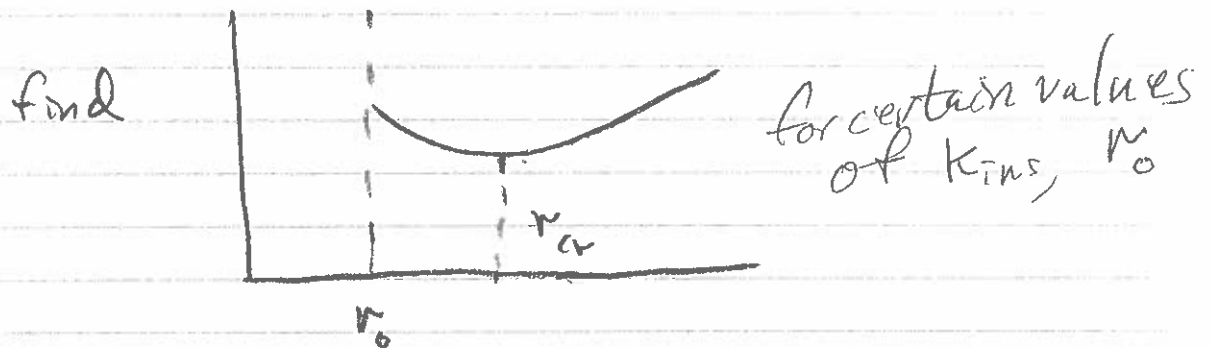
$\therefore$ Have situation



$$\therefore R = R_{o1} + R_f$$

Thus $R = \frac{\ln(r_i/r_o)}{2\pi L K_{ins}} + \frac{1}{2\pi r_i L h}$

plot R vs. r_i :



Adding insulation makes resistance go down!

Result of larger ht. trans f. surface on outside.

Find $r_{cr} = \frac{K_{ins.}}{h_f}$

if $K_{ins} < r_o h_f$, R will be monotonically increasing.

Interesting effect, can obtain same effect in spheres.

Steady S. conduction w/ ht. gen:

BS&L 9.2: conduction in a wire

Resistance dissip. gives uniform ht. gen.
(at least for DC current)

$$S_e = \frac{I^2}{K_e} \quad \begin{array}{l} \rightarrow \text{current density (amps/cm}^2\text{)} \\ \rightarrow \text{electrical conductivity} \\ \quad (\text{ohm-cm}) \end{array}$$

$$\therefore \rho \hat{C}_p \frac{\partial T}{\partial t} = K \nabla^2 T + S_e$$

Assume SS., $\therefore \frac{\partial T}{\partial t} = 0$ $\nwarrow$ assumes const. properties

by symmetry, $f^n(r)$ only.

$$\frac{K}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) + \frac{I^2}{K_e} = 0$$

$$\therefore r \frac{dT}{dr} = - \frac{I^2}{K_e K} \frac{r^2}{2} + C_1$$

$$\text{or } \frac{dT}{dr} = - \frac{I^2}{K_e K} \frac{r}{2} + \frac{C_1}{r} \quad \begin{array}{l} \nearrow C_1 = 0 \text{ since} \\ \frac{dT}{dr} = \text{finite} \\ \text{at } r=0 \end{array}$$

$$\therefore T = C_2 - \frac{I^2 r^2}{4 K_e K}$$

We need B.C., let $T = T_0$ at $r = R$

$$\therefore T - T_0 = \frac{I^2 R^2}{4k_e k} \left(1 - \frac{r^2}{R^2}\right)$$

(similar to Poiseuille flow in tube!)

What is energy flux?

$$q_r = -k \frac{\partial T}{\partial r} = \frac{I^2}{2k_e} r$$

$$\therefore Q_R = 2\pi R L \cdot \frac{I^2 R}{2k_e} = \underbrace{\pi R^2 L}_{\text{volume}} \cdot \underbrace{S_e}_{\text{source/unit volume}}$$

Let's generalize this result

consider surface ∂D enclosing domain D



At S.S., what is net energy transferred ~~from D~~ ^{from D} to surrounding solid? ~~(reverse)~~

$$\dot{Q} = \int_{\partial D} -k \nabla T \cdot \underline{\underline{n}} \, dA$$

$$= \int_D -\nabla \cdot (k \nabla T) \, dV = \int_D \dot{S}_{th} \, dV$$

from eq'n of energy

In above prob. we assumed const. k, k_e
 How do we treat prob. if $k, k_e = f^u(T)$?

We have $\nabla \cdot \underline{q} = S_{th} = k_e \frac{E^2}{L^2}$ voltage drop

$$\underline{q} = -k \nabla T \therefore \nabla \cdot (k \nabla T) = -S_e$$

w/ B.C. $T = T_0$ at $r = R$, $\frac{\partial T}{\partial r} = \text{finite}$ at $r = 0$

Problem is non-linear, $\therefore$ very difficult to solve!

Let's render dimensionless

$$\text{Let } k \Big|_{T=T_0} = k_0, \quad k_e \Big|_{T=T_0} = k_{e0}$$

$$\text{and let } \beta = \frac{r}{R}, \quad \Theta = \frac{T - T_0}{T_0}$$

$$\therefore -\frac{1}{\beta} \frac{\partial}{\partial \beta} \left[\frac{k}{k_0} \beta \frac{\partial \Theta}{\partial \beta} \right] = B \frac{k_e}{k_{e0}}$$

$$\text{Where } B \equiv \frac{k_{e0}}{k_0} \frac{R^2 E^2}{L^2 T_0}$$

To obtain sol'n, we will perform regular perturbation analysis, assuming $B \ll 1$

Since we have B.C. $\Phi|_{y=1} = 0$, we recognize that to leading order,

$$\Phi \sim B \quad \rightarrow \text{slightly different from form in book}$$

$$\therefore \text{Let } \Phi = B \Phi_0 + B^2 \Phi_1 + \dots$$

and expand $\frac{K}{K_0}$, $\frac{K_e}{K_0}$ in a Taylor series:

$$\frac{K}{K_0} = 1 - \alpha_1 \Phi - \alpha_2 \Phi^2 - \dots$$

$$\frac{K_e}{K_{e0}} = 1 - \beta_1 \Phi - \beta_2 \Phi^2 - \dots$$

End 3

Now substituting in:

$$-\frac{1}{3} \frac{d}{dz} \left[(1 - \alpha_1 (B \Phi_0 + B^2 \Phi_1 + \dots) - \alpha_2 (B \Phi_0 + \dots)^2 - \dots) \cdot 3 \frac{d}{dz} (B \Phi_0 + B^2 \Phi_1 + B^3 \Phi_2 + \dots) \right]$$

$$= B \left(1 - \beta_1 (B \Phi_0 + B^2 \Phi_1 + \dots) - \beta_2 (B \Phi_0 + \dots)^2 - \dots \right)$$

which is a mess.

Let's compare like order terms:

To $O(B)$:

$$-\frac{1}{3} \frac{d}{dz} \left[z \frac{d \cancel{H_0}}{dz} \right] = \cancel{B}$$

$$\text{w/ B.C. } H_0|_{z=1} = 0$$

$O(B^2)$:

$$-\frac{1}{3} \frac{d}{dz} \left[(-\kappa_1 H_0) z \frac{d H_0}{dz} + z \frac{d H_1}{dz} \right] = -\beta_1 H_0$$

$$\text{or } -\frac{1}{3} \frac{d}{dz} \left[z \frac{d H_1}{dz} \right] = -\beta_1 H_0 + \frac{1}{3} \frac{d}{dz} \left[\kappa_1 H_0 z \frac{d E}{dz} \right]$$

linear in H_1

$$\text{w/ B.C. } H_1|_{z=1} = 0$$

only in terms of H_0 , so known

Now for $O(B^3)$:

$$-\frac{1}{3} \frac{d}{dz} \left[z \frac{d H_2}{dz} \right] = -\beta_1 H_1 - \beta_2 H_0^2$$

(cont. next pg)

$$-\frac{1}{2} \frac{d}{dz} \left[\alpha_1 \left(H_0 z \frac{dH_1}{dz} + H_1 z \frac{dH_0}{dz} \right) + \alpha_2 H_0^2 z \frac{dH_0}{dz} \right]$$

w/ B.C. $H_2|_{z=1} = 0 \Rightarrow$ increasingly complex

Let's obtain first 2 terms:

$$-\frac{1}{2} \frac{d}{dz} \left[z \frac{dH_0}{dz} \right] = 1, \quad H_0|_{z=1} = 0$$

$$\therefore H_0 = \frac{1}{4} (1 - z^2) \text{ from previous problem}$$

Now for 2nd term:

$$\begin{aligned} -\frac{1}{2} \frac{d}{dz} \left[z \frac{dH_1}{dz} \right] &= -\beta_1 H_0 - \frac{1}{2} \frac{d}{dz} \left[\alpha_1 H_0 z \frac{dH_0}{dz} \right] \\ &= -\frac{\beta_1}{4} (1 - z^2) - \frac{1}{2} \frac{d}{dz} \left[\frac{\alpha_1}{4} (1 - z^2) z \frac{d}{dz} \left(\frac{1}{4} (1 - z^2) \right) \right] \end{aligned}$$

Integrating once:

$$z \frac{d(H)_1}{dz} = C_1 + \frac{\beta_1}{4} \left(\frac{1}{2} z^2 - \frac{1}{4} z^4 \right) + \left[\frac{\alpha_1}{4} (1 - z^2) \left(-\frac{z^2}{2} \right) \right]$$

$$\text{or } \frac{d(H)_1}{dz} = \cancel{\frac{C_1}{z}} + \frac{\beta_1}{4} z \left(\frac{1}{2} - \frac{1}{4} z^2 \right)$$

as must be finite for $z=0$ $-\frac{\alpha_1}{8} z (1 - z^2)$

Thus:

$$(H)_1 = C_2 + \frac{\beta_1}{4} \left(\frac{1}{4} z^2 - \frac{1}{16} z^4 \right) - \frac{\alpha_1}{8} \left(\frac{1}{2} z^2 - \frac{1}{4} z^4 \right)$$

Now $(H)_1 = 0$ at $z=1$

$$\therefore 0 = C_2 + \frac{\beta_1}{16} \frac{3}{4} - \frac{\alpha_1}{32}$$

$$\text{Thus } C_2 = -\frac{\beta_1 3}{64} + \frac{\alpha_1}{32}$$

And, after rearranging, we obtain:

$$\textcircled{H}_1 = \frac{1}{4}(1-z^2) \left(-\frac{\beta_1}{16}(3-z^2) + \frac{\kappa_1}{8}(1-z^2) \right)$$

or

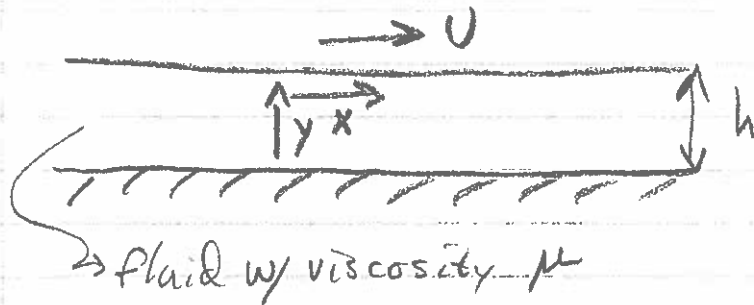
$$\textcircled{H} = \frac{\beta}{4}(1-z^2) \left[1 + \beta \left[-\frac{\beta_1}{16}(3-z^2) + \frac{\kappa_1}{8}(1-z^2) \right] + o(\beta^2) \right]$$

This is a regular perturbation expansion $\Rightarrow$

Does not always work this well. Often, can't satisfy all B.C.'s at higher order $\Rightarrow$ results in singular perturbation.

Often approach this type of problem using matched asymptotic expansion $\Rightarrow$ will discuss later.

Another regular perturbation problem:
viscous heating in bearings



what is effect of viscous heating?

we have the equation of momentum:

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla P + \nabla \cdot \underline{\underline{\tau}} + \rho \mathbf{g}$$

Assume S.S., $\mathbf{u} \equiv u \hat{e}_x$, no variation in x , z dir.

$$\therefore \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) = 0 \quad \text{w/} \quad u|_{y=0} = 0, \quad u|_{y=h} = U$$

And eq'n of energy:

$$\rho \hat{C}_p \frac{DT}{Dt} = \nabla \cdot (\kappa \nabla T) + \mu \Phi_v$$

(assume newtonian, incompressible)

$$\therefore \Phi_v = \left(\frac{\partial u}{\partial y} \right)^2$$

Thus:

$$0 = \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

$$\text{w/ } T = T_0 \text{ at } y=0, y=h$$

Now if $\mu, k \neq f^n(T)$, problem is uncoupled:

Solve momentum transp. eq'n, then E eq'n.

$$\text{i.e., } u = U \frac{y}{h}, \quad T = T_0 + \frac{1}{2} \frac{\mu U^2}{k} \left(\frac{y}{h} - \frac{y^2}{h^2} \right)$$

but we are interested in effect of non-constant properties!

Thus we obtain:

$$\frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right) = -\mu \left(\frac{\partial u}{\partial y} \right)^2 \quad T \Big|_{y=0, y=h} = T_0$$

$$\frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) = 0 \quad u \Big|_{y=0} = 0 \quad u \Big|_{y=h} = U$$

Let's render problem dimensionless

$$\text{let } \mu \Big|_{T_0} = \mu_0, \quad k \Big|_{T_0} = k_0, \quad y^* = \frac{y}{h}, \quad T^* = \frac{T - T_0}{\frac{\mu_0 U^2}{k_0}}$$

and $u^* = \frac{u}{U}$

char ΔT

Thus:

$$\frac{d}{dy^*} \left(\frac{\kappa}{\kappa_0} \frac{dT^*}{dy^*} \right) = -\frac{\mu}{\mu_0} \left(\frac{du^*}{dy^*} \right)^2; \quad T^*|_{y^*=0,1} = 0$$

and $\frac{d}{dy^*} \left(\frac{\mu}{\mu_0} \frac{du^*}{dy^*} \right) = 0; \quad u^*|_{y^*=0} = 0, \quad u^*|_{y^*=1} = 1$

End 4 we have a coupled non-linear problem!

Let's solve using a regular perturbation approach.

$$\text{Let } u^* = u^0 + N u^1 + N^2 u^2 + \dots$$

$$\text{Where } N = \frac{\mu_0 U^2}{\kappa_0 T_0} \ll 1$$

↗ dimensionless char. ΔT

$$\text{and } T^* = T^0 + N T^1 + N^2 T^2 + \dots$$

we also need a Taylor series expansion for $\frac{\mu}{\mu_0}, \frac{\kappa}{\kappa_0}$

$$\therefore \frac{\mu}{\mu_0} = 1 + T_0 \frac{\mu'}{\mu_0} (T - T_0) + \dots$$

$$= 1 + N T_0 \frac{\mu'}{\mu_0} T^* + O(N^2)$$

and

$$\begin{aligned} \frac{K}{K_0} &= 1 + N T_0 \frac{K'}{K_0} T^* + O(N^2) \\ &= 1 + N T_0 \frac{K'}{K_0} T^0 + O(N^2) \end{aligned}$$

End 4'88

Now we have the problem:

$$O(0): \frac{\partial}{\partial y^*} \left(\frac{\partial T^0}{\partial y^*} \right) = - \left(\frac{\partial u^0}{\partial y^*} \right)^2; \quad T^0 \Big|_{y^*=0,1} = 0$$

$$\text{and } \frac{\partial}{\partial y^*} \left(\frac{\partial u^0}{\partial y^*} \right) = 0; \quad u^0 \Big|_{y^*=0} = 0, \quad u^0 \Big|_{y^*=1} = 1$$

which has the solution:

$$u^0 = y^*, \quad T^0 = \frac{1}{2} (y^* - y^{*2})$$

and to next order:

$$O(1): \frac{\partial}{\partial y^*} \left(T_0 \frac{K'}{K_0} T^0 \frac{\partial T^0}{\partial y^*} + \frac{\partial T^1}{\partial y^*} \right) = - T_0 \frac{\mu'}{\mu_0} T^0 - 2 \left(\frac{\partial u^0}{\partial y^*} \right) \left(\frac{\partial u^1}{\partial y^*} \right)$$

$$\text{w/ B.C.'s: } T^1 \Big|_{y^*=0,1} = 0$$

And:

$$\frac{d}{dy^*} \left(T_0 \frac{\mu'}{\mu_0} T^0 \frac{du^0}{dy^*} + \frac{du^1}{dy^*} \right) = 0$$

w/ B.C.'s $u' \Big|_{y^*=0,1} = 0$ — note: zero at $y^*=1$!

Problem is now uncoupled: may solve momentum first, then solve for temp.

Simple matter to integrate & obtain sol'n.

Solving for u' :

$$T_0 \frac{\mu'}{\mu_0} \frac{1}{2} (y^* - y^{*2}) + \frac{du^1}{dy^*} = C_1$$

$$\therefore u' = C_1 y^* + C_2 - T_0 \frac{\mu'}{\mu_0} \frac{1}{2} \left(\frac{1}{2} y^{*2} - \frac{1}{3} y^{*3} \right)$$

$$u' \Big|_{y^*=0} = 0 \quad \therefore C_2 = 0$$

$$u' \Big|_{y^*=1} = 0 = C_1 - T_0 \frac{\mu'}{\mu_0} \frac{1}{2} \left(\frac{1}{2} - \frac{1}{3} \right)$$

$$\begin{aligned} \text{Thus } u' &= T_0 \frac{\mu'}{\mu_0} \frac{1}{2} \left(\frac{1}{6} y^* - \frac{1}{2} y^{*2} + \frac{1}{3} y^{*3} \right) \\ &= T_0 \frac{\mu'}{\mu_0} \frac{1}{12} y^* (1 - y^*) (2 - y^*) \end{aligned}$$

The observed shear stress is thus:

$$\begin{aligned}
 \tau_{yx} &= \mu \frac{\partial u}{\partial y} = \mu_0 \left(1 + T_0 N \frac{\mu'}{\mu_0} T^0 + O(N^2) \right) \\
 &\quad \cdot \left(\frac{\partial u^0}{\partial y} + N \frac{\partial u'}{\partial y} + O(N^2) \right) \\
 &= \mu_0 \left[\frac{\partial u^0}{\partial y} + N \left(T_0 \frac{\mu'}{\mu_0} T^0 \frac{\partial u^0}{\partial y} + \frac{\partial u'}{\partial y} \right) + O(N^2) \right] \\
 &= \frac{\mu_0 U}{h} \left[1 + N \left(T_0 \frac{\mu'}{\mu_0} \frac{1}{2} (y^{*2} - y^{*2}) \right) \right. \\
 &\quad \left. + T_0 \frac{\mu'}{\mu_0} \frac{1}{2} (1 - 6y^{*2} + 6y^{*2}) \right) \Big] \\
 &= \frac{\mu_0 U}{h} \left[1 + \frac{1}{2} N \frac{\partial \ln \mu}{\partial \ln T} \Big|_{T_0} + O(N^2) \right]
 \end{aligned}$$

Note: indep. of y^* as it should be

plug in #'s : (My Rheo)

1" shaft at 3600 rpm $\Rightarrow U = 480$ cm/sec

heavy machine oil: $\mu_0 = 3.3$ p, $\frac{\partial \ln \mu}{\partial \ln T} = -22 \Big|_{25^\circ\text{C}}$

$K_0 = 1.4 \times 10^4$ erg/sec cm² K $T_0 = 300^\circ\text{K}$

Thus $N = .18$

$$\text{and } T_{\max} = T_0 + \frac{N}{8} T_0 + O(N^2) \\ = T_0 + 6.8^\circ \text{K} + O(N^2)$$

with a torque reduction of

$$\frac{1}{12} N \left. \frac{\partial \ln \mu}{\partial \ln T} \right|_{T_0} = \underline{-33\% !}$$

so small ΔT 's can have lge. effects!

Now look at Convective energy transp.

Prob. 10.1, BS&L: Transpirational Cooling

Increased efficiency LOX container:

